A MESHFREE NODAL INTEGRATION METHOD FOR ELASTIC AND ELASTOPLASTIC APPLICATIONS USING THE VIRTUAL ELEMENT DECOMPOSITION



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Outline

- Motivation
- Meshfree Basis Functions
- Nodal Integration Using the Virtual Element Decomposition (NIVED)
- Numerical Tests
- Summary and Outlook

Motivation



Numerical integration of the stiffness matrix

Gauss integration

- Displacements at nodes
- Stresses/Strains at Gauss points



Nodal integration

- Displacements at nodes
- Stresses/Strains at nodes

Chen et al., IJNME, 50(2), 2001 - IJNME, 53(12), 2002; Puso et al., IJNME, 74(3), 2008:

- Direct integration at nodes leads to instabilities
- A penalty stabilization is added to the stiffness matrix

Meshfree Basis Functions



- Less sensitive to mesh distortion than FEM basis functions
- Nonpolynomial basis functions
- Linear approximation plus some nonpolynomial terms
- Integration errors affecting consistency and stability

Belytschko et al., IJNME, 37(2), 1994; Liu et al., IJNMF, 20(8-9), 1995; Atluri & Zhu, CM, 22(2), 1998; De & Bathe, CM, 25(4), 2000; Sukumar et al., IJNME, 43(5), 1998; Sukumar, IJNME, 61(12), 2004; Arroyo & Ortiz, IJNME, 65(13), 2006 ... and many others!

Linear NIVED: Bilinear Form

We want to evaluate the bilinear form of the linear elasticity boundary value problem,

$$a_E(\boldsymbol{u}^h, \boldsymbol{v}^h) = \int_E \boldsymbol{\sigma}(\boldsymbol{u}^h) : \boldsymbol{\nabla} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x}$$

at the node x_E of a representative nodal cell using a nodal projection operator Π_E and a stabilization term (both to be defined), which gives



$$a_E(\boldsymbol{u}^h, \boldsymbol{v}^h) = |E|\boldsymbol{\sigma}(\Pi_E \boldsymbol{u}^h) : \boldsymbol{\nabla}\Pi_E \boldsymbol{v}^h + \text{stabilization}$$

Linear NIVED: Virtual Element Decomposition

We use the virtual element decomposition (BdV et al., M3A, 24(8), 2014):

$$a_E^h(\boldsymbol{u}^h, \boldsymbol{v}^h) := a_E(\Pi_E \boldsymbol{u}^h, \Pi_E \boldsymbol{v}^h) + s_E(\boldsymbol{u}^h - \Pi_E \boldsymbol{u}^h, \boldsymbol{v}^h - \Pi_E \boldsymbol{v}^h)$$

where the first term gives consistency and second one stability, and Π_E is a nodal projection operator that for linear displacements is given by the "intrinsic" formula (BdV et al., M3A, 24(8), 2014):

$$\Pi_E \boldsymbol{v}^h = \left(\frac{1}{|E|} \int_E \boldsymbol{\nabla} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x}\right) \cdot (\boldsymbol{x} - \boldsymbol{x}_E) + \boldsymbol{v}^h(\boldsymbol{x}_E)$$



Linear NIVED: Virtual Element Decomp. (Cont'd)

Everything reduces to the computation of the constant strain over the element:

$$\frac{1}{|E|} \int_E \boldsymbol{\nabla} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x} = \frac{1}{|E|} \int_E \boldsymbol{\nabla}_{\mathrm{S}} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x} + \frac{1}{|E|} \int_E \boldsymbol{\nabla}_{\mathrm{AS}} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x}$$

By applying the divergence theorem, these integrals are evaluated on the boundary of the cell (def.: <u>nodal strains</u>):

$$oldsymbol{arepsilon}_{E}^{h} = rac{1}{|E|} \int_{E} oldsymbol{
abla}_{\mathrm{S}} oldsymbol{v}^{h} \, \mathrm{d}oldsymbol{x} = rac{1}{2|E|} \int_{\partial E} \left(oldsymbol{v}^{h} \otimes oldsymbol{n} + oldsymbol{n} \otimes oldsymbol{v}^{h}
ight) \, \mathrm{d}s$$
 $oldsymbol{\omega}_{E}^{h} = rac{1}{|E|} \int_{E} oldsymbol{
abla}_{\mathrm{AS}} oldsymbol{v}^{h} \, \mathrm{d}oldsymbol{x} = rac{1}{2|E|} \int_{\partial E} \left(oldsymbol{v}^{h} \otimes oldsymbol{n} - oldsymbol{n} \otimes oldsymbol{v}^{h}
ight) \, \mathrm{d}s$



Linear NIVED: Cell Stiffness Matrix

Virtual element decomposition (VED):

$$a_E^h(\boldsymbol{u}^h, \boldsymbol{v}^h) := a_E(\Pi_E \boldsymbol{u}^h, \Pi_E \boldsymbol{v}^h) + s_E(\boldsymbol{u}^h - \Pi_E \boldsymbol{u}^h, \boldsymbol{v}^h - \Pi_E \boldsymbol{v}^h)$$

consistency

stability

Nodal integration: sampling the consistency part at x_E by using the nodal projection operator leads to

$$a_E(\Pi_E \boldsymbol{u}^h, \Pi_E \boldsymbol{v}^h) = \boldsymbol{\sigma}(\Pi_E \boldsymbol{u}^h) : \boldsymbol{\nabla}\Pi_E \boldsymbol{v}^h |E|$$
$$= \left(\frac{1}{|E|} \int_E \boldsymbol{\nabla}_{\mathrm{S}} \boldsymbol{v}^h \, \mathrm{d} \boldsymbol{x}\right) : \boldsymbol{D} : \left(\frac{1}{|E|} \int_E \boldsymbol{\nabla}_{\mathrm{S}} \boldsymbol{u}^h \, \mathrm{d} \boldsymbol{x}\right) |E|$$
$$= \boldsymbol{q}^{\mathsf{T}} \, \boldsymbol{K}_E^{\mathrm{c}} \, \boldsymbol{d} \quad \text{(consistency stiffness)}$$

Linear NIVED: Cell Stiffness Matrix

Virtual element decomposition (VED):

$$a_E^h(\boldsymbol{u}^h, \boldsymbol{v}^h) := a_E(\Pi_E \boldsymbol{u}^h, \Pi_E \boldsymbol{v}^h) + s_E(\boldsymbol{u}^h - \Pi_E \boldsymbol{u}^h, \boldsymbol{v}^h - \Pi_E \boldsymbol{v}^h)$$

consistency

stability

For the stability part the approximate s_E is chosen in the spirit of the "*D*-recipe" (BdV et al., CMA, 74(5), 2017):

 $s_E(\boldsymbol{u}^h - \Pi_E \boldsymbol{u}^h, \boldsymbol{v}^h - \Pi_E \boldsymbol{v}^h) = \boldsymbol{q}^{\mathsf{T}}(\boldsymbol{I} - \Pi_E) \operatorname{diag}(\operatorname{diag}(\boldsymbol{K}_E^c)) (\boldsymbol{I} - \Pi_E) \boldsymbol{d}$ $= \boldsymbol{q}^{\mathsf{T}} \boldsymbol{K}_E^s \boldsymbol{d} \quad \text{(stability stiffness)}$

Nonlinear NIVED: Incremental Solution

VED representation of the Virtual Work: given $\alpha_n = \alpha(t_n)$ find u_{n+1}^h such that

$$|E| \hat{\boldsymbol{\sigma}}_{E} (\boldsymbol{\alpha}_{n}, (\boldsymbol{\varepsilon}_{E}^{h})_{n+1}) : \delta \boldsymbol{\varepsilon}_{E}^{h} + \int_{E} \hat{\boldsymbol{\sigma}}_{E} (\boldsymbol{\alpha}_{n}, (\boldsymbol{\varepsilon}^{h} - \boldsymbol{\varepsilon}_{E}^{h})_{n+1}) : \delta (\boldsymbol{\varepsilon}^{h} - \boldsymbol{\varepsilon}_{E}^{h}) d\boldsymbol{x} - |E| \boldsymbol{b}_{n+1}^{h} (\boldsymbol{x}_{E}) \cdot \delta \boldsymbol{u}^{h} - |S| \, \bar{\boldsymbol{t}}_{n+1} (\boldsymbol{x}_{S}) \cdot \delta \boldsymbol{u}^{h} = 0$$

where x_S is a node on the Neumann boundary.

Linearization using the small strain assumption:

$$\begin{aligned} (\delta \boldsymbol{\varepsilon}_{E}^{h})^{\mathsf{T}} \boldsymbol{D}_{t} \Delta \boldsymbol{\varepsilon}_{E}^{h} \left| E \right| + (\boldsymbol{I} - \boldsymbol{\Pi}_{E})^{\mathsf{T}} \boldsymbol{s}_{E} (\Delta \boldsymbol{u}_{n+1}^{h}, \delta \boldsymbol{u}^{h}) (\boldsymbol{I} - \boldsymbol{\Pi}_{E}) \\ &= - \left\{ (\delta \boldsymbol{\varepsilon}_{E}^{h})^{\mathsf{T}} (\boldsymbol{\sigma}_{E})_{n+1} |E| + (\boldsymbol{I} - \boldsymbol{\Pi}_{E})^{\mathsf{T}} \boldsymbol{s}_{E} (\boldsymbol{u}_{n+1}^{h}, \delta \boldsymbol{u}^{h}) (\boldsymbol{I} - \boldsymbol{\Pi}_{E}) \\ &- (\delta \boldsymbol{u}^{h})^{\mathsf{T}} \boldsymbol{b}_{n+1}^{h} (\boldsymbol{x}_{E}) |E| \\ &- (\delta \boldsymbol{u}^{h})^{\mathsf{T}} \bar{\boldsymbol{t}}_{n+1} (\boldsymbol{x}_{S}) |E| \right\} \end{aligned}$$

Numerical Tests (Linear NIVED): Integration Cells



Meshes use the same node set



NIVED 1-pt per edge (basis function only)

MEM

Internal points: 1-pt, 3-pt, 6-pt, 12-pt (basis functions and derivatives)

Linear NIVED: Patch Test





Linear NIVED: Manufactured Problem

Convergence



Linear NIVED: Manufactured Problem (Cont'd)

Timing



Normalized CPU time = $T/T_{\rm max}$

Nonlinear NIVED: Perforated Plate



Dimensions:

- H = 180 mm
- L = 100 mm
- R = 50 mm

Material:

- J2-Plasticity / Plane strain
- Huber-Mises-Hencky yield function
- Mixed isotropic and kinematic hardening
- E = 70 MPa
- v = 0.2
- H_i = 10 MPa
- H_k = 10 MPa
- σ_{y0} = 0.8 MPa

Top displacement:

• δ = 10 mm

Nonlinear NIVED: Perforated Plate (Cont'd)



Nonlinear NIVED: Perforated Plate (Cont'd)



Displacement solution:

	NIVED (2624 dofs)	NIVED (6314 dofs)	NIVED (13458 dofs)	FEM-Q4 (49284 dofs)
u _y (B)	8.789	8.791	8.792	8.792
u _x (A)	-4.584	-4.597	-4.606	-4.609

Nonlinear NIVED: Locking Effect



Dimensions:

- H = 180 mm
- L = 100 mm
- R = 50 mm

Material:

- J2-Plasticity / Plane strain
- Huber-Mises-Hencky yield function
- Perfect plasticity
- E = 7000 kgf/mm2
- v = 0.3
- H_i = 0 MPa
- H_k = 0 MPa
- $\sigma_{y0} = 24.3 \text{ kgf/mm2}$

Top displacement:

• δ = 2 mm

Nonlinear NIVED: Locking Effect (Cont'd)



(*) Zienkiewicz & Taylor, The Finite Element Method for Solid and Structural Mechanics, Elsevier Butterworth-Heinemann, Sixth Edition, 2005.

Summary and Outlook

- Developed NIVED (Nodal Integration VED) for elastic and elastoplastic applications.
- VED provides consistency (patch test satisfaction) and optimal convergence to the nodal integration method.
- Computational cost of NIVED is similar to the computational cost of a 3-pt standard Gauss rule.
- Outlook: locking effect, 3D simulations, large deformations.

Journal Acronyms

- M3A: Mathematical Models and Methods in Applied Sciences
- CMAME: Computer Methods in Applied Mechanics and Engineering
- IJNME: International Journal for Numerical Methods in Engineering
- IJNMF: International Journal for Numerical Methods in Fluids
- CM: Computational Mechanics
- CMA: Computers & Mathematics with Applications